

**MEEM 3700**  
**Mechanical Vibrations**

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Mechanical Engineering-Engineering Mechanics

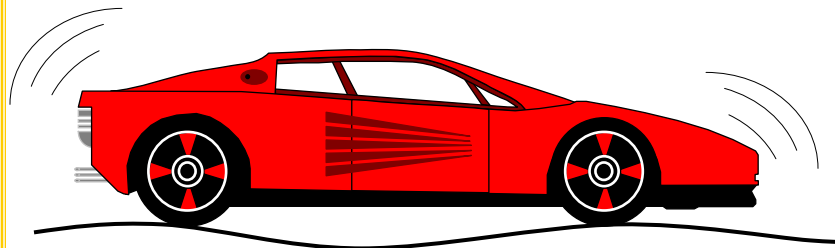
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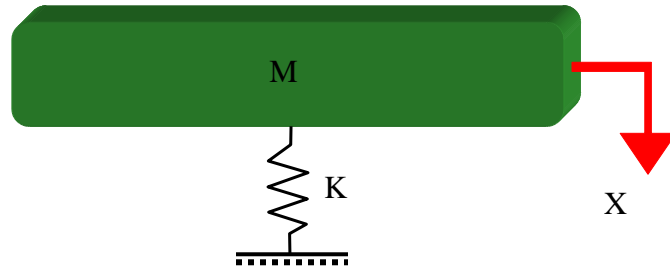
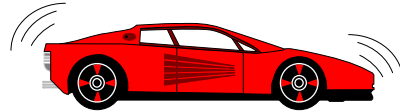
**Multiple Degree of Freedom Systems**



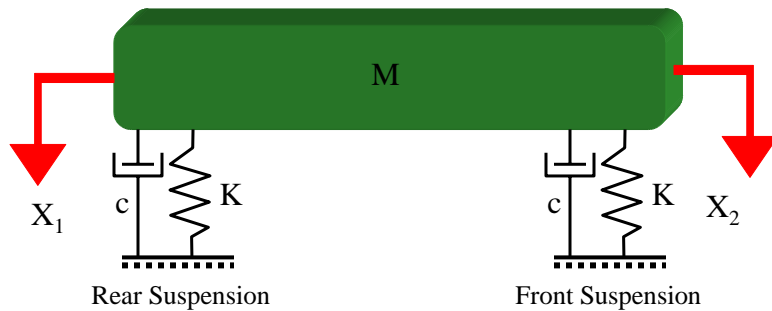
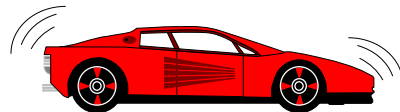
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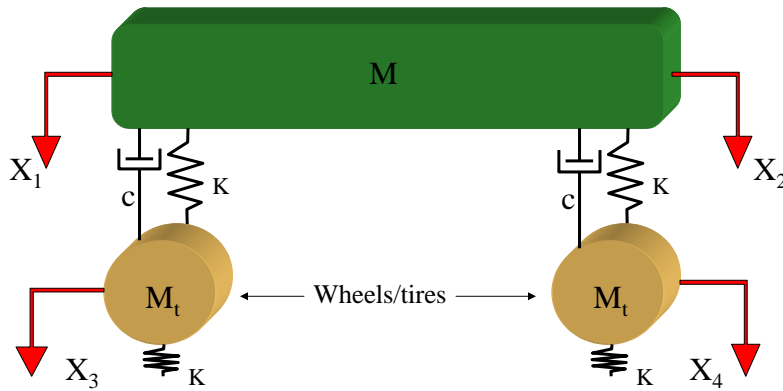
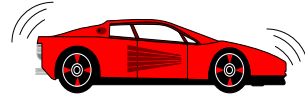
### Vibrating Systems: Single Degree of Freedom (SDOF)



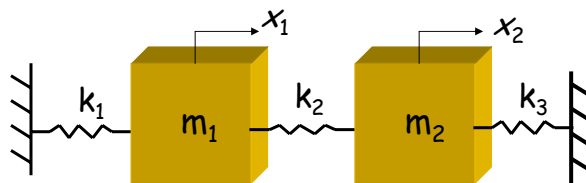
### Vibrating Systems: Multi Degree of Freedom (MDOF)



## Vibrating Systems: Multi Degree of Freedom (MDOF)



## 2 Degrees of Freedom: Free Vibration

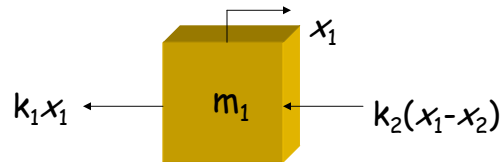


Given a set of initial conditions, determine the free vibration response of the system.

$$\begin{aligned}
 & x_1(0) \quad \dot{x}_1(0) \quad x_2(0) \quad \dot{x}_2(0) \\
 & x_1(t) = ? \quad x_2(t) = ?
 \end{aligned}$$

**2 Degrees of Freedom: Free Vibration**

F.B.D. for mass 1



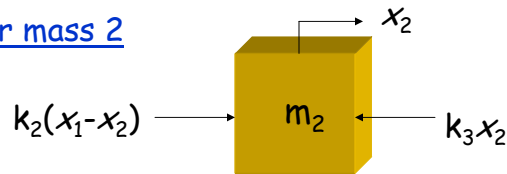
$$\sum F_{x_1} = \sum F_{eff} = m_1 \ddot{x}_1$$

$$-k_1x - k_2(x_1 - x_2) = m_1 \ddot{x}_1$$

$$m_1 \ddot{x}_1 + (k_1 + k_2)x_1 - k_2x_2 = 0 \quad \text{EOM \#1}$$

**2 Degrees of Freedom: Free Vibration**

F.B.D. for mass 2



$$\sum F_{x_2} = \sum F_{eff}$$

$$k_2(x_1 - x_2) - k_3x_2 = m_2 \ddot{x}_2$$

$$m_2 \ddot{x}_2 + (k_2 + k_3)x_2 - k_2x_1 = 0 \quad \text{EOM \#2}$$

**2 Degrees of Freedom: Free Vibration**

$$m_1 \ddot{x}_1 + (k_2 + k_1)x_1 - k_2 x_2 = 0$$

$$m_2 \ddot{x}_2 + (k_2 + k_3)x_2 - k_2 x_1 = 0$$

2<sup>nd</sup> order differential equations

Linear

Homogeneous

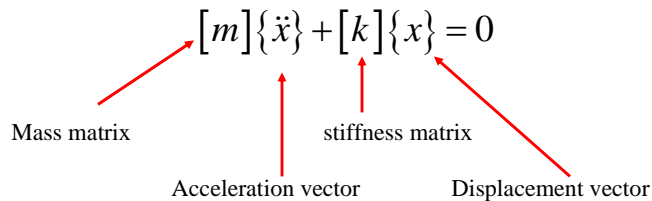
Constant coefficients

Coupled

**2 Degrees of Freedom: Free Vibration**

For a solution to simultaneous equations, rewrite in matrix form.

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$



**2 Degrees of Freedom: Free Vibration**

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

assume solution and substitute

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} \sin(\omega t + \phi)$$

$$\begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} = -\omega^2 \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} \sin(\omega t + \phi)$$

**2 Degrees of Freedom: Free Vibration**

$$-\omega^2 \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} \sin(\omega t + \phi) + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} \sin(\omega t + \phi) = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\left( -\omega^2 \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix} \right) \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} \sin(\omega t + \phi) = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\left( -\omega^2 \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix} \right) \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\begin{bmatrix} k_1 + k_2 - \omega^2 m_1 & -k_2 \\ -k_2 & k_2 + k_3 - \omega^2 m_2 \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$



A solution to this set of equations exists if the determinant of the coefficient matrix is equal to zero.

**2 Degrees of Freedom: Free Vibration**

$$\det \begin{bmatrix} k_1 + k_2 - \omega^2 m_1 & -k_2 \\ -k_2 & k_2 + k_3 - \omega^2 m_2 \end{bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$(k_1 + k_2 - \omega^2 m_1)(k_2 + k_3 - \omega^2 m_2) - k_2^2 = 0$$

let :  $\lambda = \omega^2$

$$(k_1 + k_2 - \lambda m_1)(k_2 + k_3 - \lambda m_2) - k_2^2 = 0$$

$$(k_1 + k_2)(k_2 + k_3) - \lambda m_2 (k_1 + k_2) - \lambda^2 m_1 m_2 - k_2^2 = 0$$

$$m_1 m_2 \lambda^2 - \lambda [m_2 (k_1 + k_2) + m_1 (k_2 + k_3)] + (k_1 + k_2)(k_2 + k_3) - k_2^2 = 0$$

**2 Degrees of Freedom: Free Vibration**

$$m_1 m_2 \lambda^2 - \lambda [m_2 (k_1 + k_2) + m_1 (k_2 + k_3)] + (k_1 + k_2)(k_2 + k_3) - k_2^2 = 0$$

A quadratic equation in  $\lambda$

$$\lambda_{1,2} = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a}$$

$$\lambda_{1,2} = \frac{-[m_2 (k_1 + k_2) + m_1 (k_2 + k_3)] \pm \sqrt{[m_2 (k_1 + k_2) + m_1 (k_2 + k_3)]^2 - 4m_1 m_2 ((k_1 + k_2)(k_2 + k_3) - k_2^2)}}{2m_1 m_2}$$

Recall  $\longrightarrow \lambda = \omega^2$

Therefore  $\longrightarrow \omega_1 = \sqrt{\lambda_1} \quad \omega_2 = \sqrt{\lambda_2}$

$\omega_1$  and  $\omega_2$  are the natural frequencies of the system

**2 Degrees of Freedom: Free Vibration**

$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} \sin(\omega t + \phi)$$

for each value of  $\omega_1$  and  $\omega_2$  there is a corresponding vector  $\begin{Bmatrix} X_1^1 \\ X_2^1 \end{Bmatrix}$   $\begin{Bmatrix} X_1^2 \\ X_2^2 \end{Bmatrix}$

$\begin{Bmatrix} X_1^1 \\ X_2^1 \end{Bmatrix}$   $\begin{Bmatrix} X_1^2 \\ X_2^2 \end{Bmatrix}$  are called the mode shapes of the system

The total solution is:

$$\begin{Bmatrix} x_1(t) \\ x_2(t) \end{Bmatrix} = A \begin{Bmatrix} X_1^1 \\ X_2^1 \end{Bmatrix} \sin(\omega_1 t + \phi_1) + B \begin{Bmatrix} X_1^2 \\ X_2^2 \end{Bmatrix} \sin(\omega_2 t + \phi_2)$$

**2 Degrees of Freedom: Free Vibration**

Alternative Solution method: Eigen-Solution

$$\left( -\omega^2 \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix} \right) \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} = \omega^2 \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix}$$

Let  $\lambda = \omega^2$

$$\begin{matrix} \rightarrow \\ \rightarrow \end{matrix} [k] \{x\} = \lambda [m] \{x\}$$

Eigenvalue problem

**2 Degrees of Freedom: Free Vibration**

$$[k]\{x\} = \lambda [m]\{x\}$$

eigenvalues =  $\lambda_1, \lambda_2, \lambda_3, \dots, \lambda_N$  where N = the number of degrees of freedom

and associated eigenvectors  $\begin{Bmatrix} X_1^1 \\ X_2^1 \end{Bmatrix}, \begin{Bmatrix} X_1^2 \\ X_2^2 \end{Bmatrix}, \begin{Bmatrix} X_1^3 \\ X_2^3 \end{Bmatrix}, \dots, \begin{Bmatrix} X_1^N \\ X_2^N \end{Bmatrix}$

The **natural frequencies** are the square root of the eigenvalues

$$\omega_i = \sqrt{\lambda_i}$$

The eigenvectors are also called the **mode shapes of the system**