

MEEM 3700
Mechanical Vibrations

Mohan D. Rao

Chuck Van Karsen

Mechanical Engineering-Engineering Mechanics

Michigan Technological University

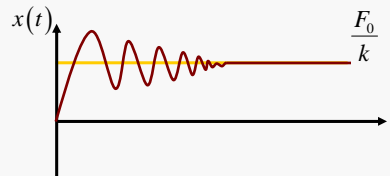
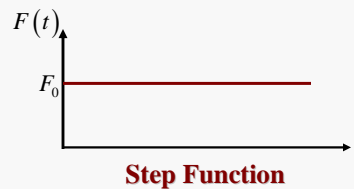
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Laplace Transform: Example 1

$$m\ddot{x} + c\dot{x} + kx = F_0$$

$$x(0) = 0$$

$$\dot{x}(0) = 0$$



Laplace Transform: Example 1

$$m\ddot{x} + c\dot{x} + kx = F_0$$

$$m(s^2 X(s) - sx(0) - \dot{x}(0)) + c(sX(s) - x(0)) + kX(s) = \frac{F_0}{s}$$

$$(ms^2 + cs + k)X(s) = \frac{F_0}{s}$$

$$X(s) = \frac{F_0}{s(ms^2 + cs + k)}$$

Laplace Transform: Example 1

$$X(s) = \frac{F_0}{s(ms^2 + cs + k)} = \frac{F_0/m}{s\left(s^2 + \frac{c}{m}s + \frac{k}{m}\right)}$$

$$X(s) = \frac{F_0/m}{s(s-\lambda)(s-\lambda^*)}$$

Partial Fraction Expansion

$$X(s) = \frac{F_0/m}{s(s-\lambda)(s-\lambda^*)} = \frac{A}{s} + \frac{B}{s-\lambda} + \frac{C}{s-\lambda^*}$$

$$x(t) = A + Be^{\lambda t} + Ce^{\lambda^* t}$$

$$\lambda = -\frac{c}{2m} \pm \frac{\sqrt{\left(\frac{c}{m}\right)^2 - 4\left(\frac{k}{m}\right)}}{2}$$

$$\lambda = -\zeta\omega_n \pm j\omega_d$$

If $\zeta < 1$

Laplace Transform: Example 1

Pole – Residue Theorem $A, B, C \rightarrow$ **Residues**

$$\frac{F_0/m}{s(s-\lambda)(s-\lambda^*)} = \frac{A}{s} + \frac{B}{s-\lambda} + \frac{C}{s-\lambda^*}$$

multiply by s

evaluate @ s = 0 (root)

$$\frac{F_0/m}{(-\lambda)(-\lambda^*)} = \frac{A\cancel{s}}{\cancel{s}} + \frac{B\cancel{s}}{s-\lambda} + \frac{C\cancel{s}}{s-\lambda^*} \quad \rightarrow \quad A = \frac{F_0/m}{\lambda\lambda^*}$$

$$\frac{F_0/m \cancel{(s-\lambda)}}{s \cancel{(s-\lambda)} (s-\lambda^*)} \Big|_{s=\lambda} = \frac{A \cancel{(s-\lambda)}}{s} + B + \frac{C \cancel{(s-\lambda)}}{s-\lambda^*} \Big|_{s=\lambda}$$

Laplace Transform: Example 1

$$\frac{F_0/m}{\lambda(\lambda-\lambda^*)} = B$$

$$A = \frac{F_0/m}{\lambda\lambda^*}$$

$$\frac{F_0/m}{\lambda^*(\lambda^*-\lambda)} = C$$

$$A = \frac{F_0}{k}$$

$$x(t) = \frac{F_0}{k} + Be^{\lambda t} + Ce^{\lambda^* t}$$

$$x(t) = \frac{F_0}{k} + B(e^{-\zeta\omega_n t} e^{j\omega_d t}) + C(e^{-\zeta\omega_n t} e^{-j\omega_d t})$$

Use $e^{\pm j\omega t} = \cos(\omega t) \pm j \sin(\omega t)$

Laplace Transform: Example 1

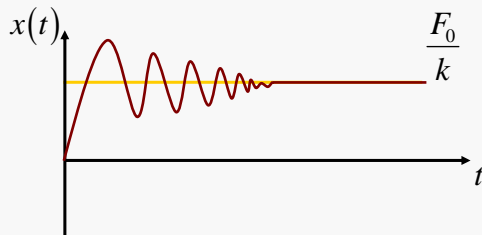
$$x(t) = \frac{F_0}{k} + e^{-\zeta\omega_n t} [B e^{j\omega_d t} + C e^{-j\omega_d t}]$$

$$x(t) = \frac{F_0}{k} + e^{-\zeta\omega_n t} [(B + C) \cos(\omega_d t) + j(B - C) \sin(\omega_d t)]$$

Substitute for B & C, rearrange & plot

$$\frac{F_0/m}{\lambda(\lambda - \lambda')} = B$$

$$\frac{F_0/m}{\lambda'(\lambda' - \lambda)} = C$$



Laplace Transform: Example 2

SDOF system subjected to a harmonic excitation

$$10\ddot{x} + \dot{x} + 2000x = 50 \cos(20t) \quad x(0) = 2, \quad \dot{x}(0) = 0$$

$$x(t) = ?$$

$$m(s^2 X(s) - 2s + 0) + c(sX(s) - 2) + kX(s) = \frac{50s}{s^2 + 20^2}$$

$$X(s) = \frac{50s}{(10s^2 + s + 2000)(s^2 + 400)} + \frac{20s + 2}{(10s^2 + s + 2000)}$$

$$X(s) = \frac{5s}{(s - \lambda)(s - \lambda^*)(s - j\omega)(s + j\omega)} + \frac{2s + 0.2}{(s - \lambda)(s - \lambda^*)}$$

$$\lambda = -0.05 + 14.14j \quad \omega = 20$$

Laplace Transform: Example 2

$$X(s) = \frac{A}{s-\lambda} + \frac{B}{s-\lambda^*} + \frac{C}{s-j\omega} + \frac{D}{s+j\omega} + \frac{E}{s-\lambda} + \frac{F}{s-\lambda^*}$$

$$B = A^* , D = C^* , F = E^*$$

$$A = \frac{5\lambda}{(\lambda-\lambda^*)(\lambda-j20)(\lambda+j20)} = 0.0125 + 0.0001j$$

$$C = \frac{5(j20)}{(20j-\lambda)(20j-\lambda^*)(20j+20j)} = -0.0125 - 0.0001j$$

$$E = \frac{2\lambda+0.2}{(\lambda-\lambda^*)} = 1.0 - 0.0035j$$

$$B = 0.0125 - 0.0001j \quad D = -0.0125 + 0.0001j \quad F = 1.0 + 0.0035j$$

Laplace Transform: Example 2

$$X(s) = \frac{A+E}{(s-\lambda)} + \frac{B+F}{(s-\lambda^*)} + \frac{C}{s-j\omega} + \frac{D}{s+j\omega}$$

$$x(t) = (A+E)e^{\lambda t} + (B+F)e^{\lambda^* t} + Ce^{j\omega t} + De^{-j\omega t}$$

$$x(t) = e^{-0.5t} \left[(A+E)e^{14.14jt} + (B+F)e^{-j14.14t} \right] + Ce^{20jt} + De^{-20jt}$$

$$e^{\pm j\omega t} = \cos(\omega t) \pm j \sin(\omega t)$$

$$x(t) = e^{-0.5t} (2.025 \cos(14.14t) + 0.007 \sin(14.14t)) + 0.025 \cos(20t - \pi)$$

This is similar to what was obtained before by the classical method

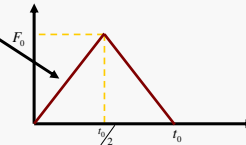
$$\mathbf{x}(t) = \mathbf{x}_{\text{transient}} + \mathbf{x}_{\text{steady state}}$$

Laplace Transform: Example 3– Triangular Pulse Excitation

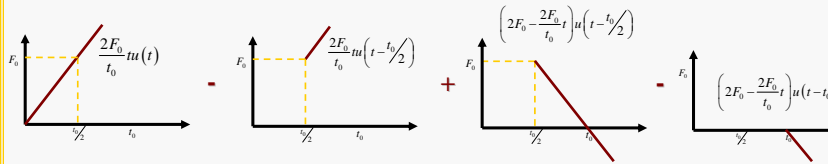
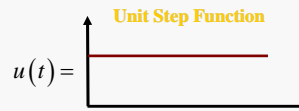
Determine the response of an undamped mass-spring system at rest in equilibrium to a symmetric triangular pulse $F(t)$ of total duration t_0

Given: $m\ddot{x} + kx = F(t)$ and $x(0) = 0, \dot{x}(0) = 0$

Find: $x(t)$



Solution: The graphical breakdown of the triangular pulse using unit step functions is shown below.



Laplace Transform: Example 3

The mathematical form of the expression is

$$F(t) = \frac{2F_0 t}{t_0} \left[u(t) - u\left(t - \frac{t_0}{2}\right) \right] + \left(\frac{-2F_0 t}{t_0} + 2F_0 \right) \left[u\left(t - \frac{t_0}{2}\right) - u(t - t_0) \right]$$

$$= \frac{2F_0 t}{t_0} u(t) - \frac{4F_0}{t_0} \left(t - \frac{t_0}{2} \right) u\left(t - \frac{t_0}{2} \right) + \frac{2F_0}{t_0} (t - t_0) u(t - t_0)$$

The Laplace transform of $F(t)$ is obtained using the **second shifting theorem** and the proper transform from the Laplace transform table.

$$F(s) = \frac{2F_0}{t_0 s^2} \left[1 - 2e^{-s \frac{t_0}{2}} + e^{-s t_0} \right]$$

The transform of the system response is

$$X(s) = \frac{F(s)}{m(s^2 + \omega_n^2)}$$

Second Shifting Theorem

If $L[f(t)] = F(s)$ and $g(t) = f(t-a)$ for $a > 0$
 $= 0$ for $t < 0$, then $L[g(t)] = e^{-as} F(s)$

Laplace Transform: Example 3

Use partial fraction decomposition to express $\frac{1}{s^2(s^2 + \omega_n^2)} = \frac{A}{s} + \frac{B}{s^2} + \frac{Cs + D}{s^2 + \omega_n^2}$

solving, $A = -C = 0,$

$B = 1/\omega_n^2, D = -B$

Hence $\frac{1}{s^2(s^2 + \omega_n^2)} = \frac{1}{\omega_n^2} \left(\frac{1}{s^2} - \frac{1}{(s^2 + \omega_n^2)} \right)$

$$X(s) = \frac{2F_0}{m\omega_n^2} \left(\frac{1}{s^2} - \frac{1}{s^2 + \omega_n^2} \right) \left(1 - 2e^{-\frac{s t_0}{2}} + e^{-s t_0} \right)$$

Laplace Transform: Example 3

Application of the **first shifting theorem** and transform pairs from the Laplace transform table are used to invert the transform and obtain the system response:

$$x(t) = \frac{2F_0}{m\omega_n^2} \left\{ \left[t - \frac{1}{\omega_n} \sin(\omega_n t) \right] u(t) - 2 \left[t - \frac{t_0}{2} - \frac{1}{\omega_n} \sin\left(\omega_n \left(t - \frac{t_0}{2}\right)\right) \right] u\left(t - \frac{t_0}{2}\right) + \left[t - t_0 - \frac{1}{\omega_n} \sin(\omega_n(t - t_0)) \right] u(t - t_0) \right\}$$

This example illustrates application of the Laplace transform to determine the response of an undamped one-degree-of-freedom system subject to an excitation whose form changes with time.

First Shifting Theorem
 If $L[x(t)] = X(s)$, then
 $L[e^{at} f(t)] = F(s-a)$

Partial Fraction Expansion with Complex Roots

$$X(s) = \frac{1}{(s^2 + a_1s + a_2)}$$

Here, the polynomial has two roots that are complex.

Solution Procedure Write: $s^2 + a_1s + a_2 = (s + a_1/2)^2 + (a_2 - a_1^2/4)$
 $= (s + a)^2 + b^2 \rightarrow a = a_1/2$ and $b = (a_2 - a_1^2/4)$

Then Write

$$\frac{1}{(s^2 + a_1s + a_2)} = \frac{A(s + a)}{(s + a)^2 + b^2} + \frac{Bb}{(s + a)^2 + b^2}$$

Obtain A and B the usual way

The Inverse Laplace Transform is given by

$$x(t) = L[X(s)] = A e^{-at} \cos bt + B e^{-at} \sin bt$$